Example 4.1 [Uni-axial Column Design]

1. Design the braced short column to sustain a design load of 1100 KN and a design moment of 160KNm which include all other effects .Use C25/30 and S460.

Take $\frac{d'}{H} = 0.1$ and section 270 mm * 450 mm



Solution

Step 1- Material

$$f_{cd} = \frac{0.85 * 25}{1.5} = 14.16667 mpa$$
$$f_{yd} = \frac{460}{1.15} = 400 mpa$$

Step 2-Determine the normalized axial and bending moment value

$$v_{sd} = \frac{N_{sd}}{f_{cd}bh} = \frac{1100 * 10^3}{14.1667 * 270 * 450} = 0.639$$
$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{160 * 10^6}{14.1667 * 270 * 450^2} = 0.2065$$

Step 3-Using $\frac{d'}{H} = 0.1$ read the mechanical steel ratio from uniaxial interaction chart for $V_{sd} = 0.639$ $\mu_{sd} = 0.2065$

From interaction chart $\omega = 0.3$

$$A_{s,tot} = \frac{\omega f_{cd} bd}{f_{yd}} = \frac{0.3 * 14.166 * 270 * 450}{400} = 1290.937 \ mm^2$$
$$A = \frac{A_{s,tot}}{2} = \frac{1290.937}{2} = 645.468 \ mm^2$$

Check with maximum and minimum reinforcement limit

$$A_{s,min} = max \begin{cases} \frac{0.1 N_{ED}}{f_{yd}} = 275 mm^2 & OK! \\ 0.002A_c & \\ A_{s,max} = 0.08 A_c = 0.08 * 270 * 450 = & OK! \end{cases}$$

Step 4- Detailing





270mm

Example 4.2 [Biaxial column design]

Determine the longitudinal reinforcement for corner column of size 400*400 mm and the design factored moment and axial force of

$$P = 1360 \text{ KN} \quad M_{sd,h} = 200 \text{ KNm} \quad M_{sd,b} = 100 \text{ KNm}$$

Use C25/30 and S460 class 1 work take $\frac{h'}{h} = \frac{b'}{h} = 0.1$

Solution

Step 1- Material

$$f_{cd} = \frac{0.85 * 25}{1.5} = 14.16667 mpa$$
$$f_{yd} = \frac{460}{1.15} = 400 mpa$$

Step 2- Determine the normalized axial and bending moment value

$$v_{sd} = \frac{p}{f_{cd}bh} = \frac{1360*10^3}{14.1667*400*400} = 0.6$$
$$\mu_{sd,h} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{200*10^6}{14.1667*400*400^2} = 0.2206$$

$$\mu_{sd,b} = \frac{m_{sd}}{f_{cd}db^2} = \frac{100 * 10^6}{14.1667 * 400 * 400^2} = 0.1103$$

Step 3- Find ω using $\frac{d'}{d} = \frac{b'}{B} = 0.1$, $V_{sd} = 0.6$, $\mu_{sd,h} = 0.2206$, $\mu_{sd,b} = 0.1103$

From biaxial chart $\omega = 0.6$

$$A_{s,tot} = \frac{\omega f_{cd} b d}{f_{yd}} = \frac{0.6 * 14.166 * 400 * 400}{400} = 3400 \ mm^2$$

Check with maximum and minimum reinforcement limit

$$A_{s,min} = max \begin{cases} \frac{0.1 N_{ED}}{f_{yd}} = 340 \ mm^2 \end{cases}$$
 OK

$$A_{s,max} = 0.08 A_c = 0.08 * 400 * 400 = 12800$$
 OK!

Step 4- Detailing



Example 4.3 [Column]

Determine whether the column CD is slender or not, if it is subjected t loads shown below. Consider the frame to be non-sway



Use C25/30 $f_{cd} = 14.1667 mpa$

Solution

Step 1- Slenderness limit

$$\lambda_{lim} = \frac{20ABC}{\sqrt{n}} take A = 0.7 B = 1.1 C = 1.7 - r_m$$

where
$$r_m = \frac{m_{01}}{m_{02}} = \frac{-34.4}{68.8} = -0.5$$
 $C = 1.7 - (-0.5) = 2.2$

$$n = \frac{N_{ed}}{A_c f_{cd}} = \frac{516 * 10^3}{14.1667 * 400 * 400} = 0.227647$$
$$\lambda_{lim} = \frac{20 * 0.7 * 1.1 * 2.2}{\sqrt{0.227647}} = 71.0088$$

Step 2-Find the slenderness of the column

2.1 Effective length

For Braced member

$$l_{0} = 0.5l \sqrt{\left[1 + \frac{K_{1}}{0.5 + K_{1}}\right] \left[1 + \frac{K_{2}}{0.5 + K_{2}}\right]}$$

$$K = \frac{Column \ stiffness}{\Sigma \ beam \ stiffness}$$

$$K_{i} = \frac{\binom{EI}{l} column}{2(2^{EI}/l) \ beam}$$

$$I_{column} = \frac{400 * 400^3}{12} = 2133333333 \, mm^4$$

$$I_{beam} = \frac{250 * 500^3}{12} = 2604166667 \ mm^4$$

$$K_1 = \frac{\frac{2133333333E}{8000}}{2(\frac{2*2604166667E}{6000})} = 0.1536$$

For fixed restraint K=0 but in reality we cannot provide a fully fixed support so use $K_2 = 0.1$

$$\begin{split} l_0 &= 0.5 * 6000 \sqrt{\left[1 + \frac{0.1536}{0.5 + 0.1536}\right] \left[1 + \frac{0.1}{0.5 + 0.1}\right]} = 3601.050 \ mm \\ \lambda &= \frac{l_o}{i} \quad i = \sqrt{\frac{I}{A}} = \sqrt{\frac{2133333333}{160000}} = 115.470 \ mm \\ \lambda &= \frac{3601.0504}{115.470} = 31.186 \ mm \\ \lambda &< \lambda_{lim} \qquad Short \ column \end{split}$$

Example 4.4 [Column Design]

Design the braced column to resist an axial load of 950 KN and a moment of M_{sd} =115 KNM at the top and M_{sd} =-95 KNM at the bottom as shown below .length of the column is 5.5 m and cross-section of 300*300 mm use C25/30 and S460 take L_e=0.66L



<u>Solution</u>

Step 1- Material

$$f_{cd} = \frac{0.85 * 25}{1.5} = 14.16667 \, mpa$$
$$f_{yd} = \frac{460}{1.15} = 400 \, mpa$$

Step 2- Check slenderness limit

$$\lambda_{lim} = \frac{20ABC}{\sqrt{n}} + take A = 0.7$$
 $B = 1.1$ $C = 1.7 - r_m$
where $r_m = \frac{m_{01}}{m_{02}} = \frac{-95}{115} = -0.826$ $C = 1.7 - (-0.826) = 2.526$

$$n = \frac{N_{ed}}{A_c f_{cd}} = \frac{950 * 10^3}{14.1667 * 300 * 300} = 0.745$$
$$\lambda_{lim} = \frac{20 * 0.7 * 1.1 * 2.526}{\sqrt{0.745}} = 45.066$$

Step 3-Slenderess

$$\lambda = \frac{l_o}{i}$$
 $i = \sqrt{\frac{I}{A}} = \sqrt{\frac{675000000}{90000}} = 86.6025 \, mm$

$$\begin{split} \lambda &= \frac{0.66 * 5500}{86.6025} = 41.915 \\ \lambda &< \lambda_{lim} \quad Short \ column \ negelect \ second \ order \ effect \end{split}$$

Step 4- accidental eccentricity

$$e_a = \frac{l_o}{400} = \frac{3630}{400} = 9.075 \ mm$$

Step 5- Equivalent first order eccentricity

$$e_{e} = max \begin{cases} 0.6e_{02} + 0.4 \ e_{01} \\ 0.4e_{02} \end{cases}$$

$$e_{02} = \frac{M_{02}}{N_{sd}} = \frac{115 * 10^{6}}{950 * 10^{3}} = 121.052 \ mm \end{cases}$$

$$e_{01} = \frac{M_{01}}{N_{sd}} = \frac{95 * 10^{6}}{950 * 10^{3}} = 100 \ mm \end{cases}$$

$$e_{e} = max \begin{cases} 0.6e_{02} + 0.4 \ e_{01} \\ 0.4e_{02} \end{cases} = 48.4208 \ mm \end{cases}$$

 $e_{tot} = e_o + e_e + e_2 = 9.075 + 48.4208 + 0 = 57.4958 mm$ check with $e = e_{02} + e_a = 121.052 + 9.075 = 130.127 mm$ So take $e_{tot} = 130.127 mm$

Step 6- Design

 $N_{sd} = 950 \ KN$ $M_{sd} = N_{sd} * e_{tot} = 123.62 \ KNm$

$$v_{sd} = \frac{p}{f_{cd}bh} = \frac{950 * 10^3}{14.1667 * 300 * 300} = 0.745$$
$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{123.62 * 10^6}{14.1667 * 300 * 300^2} = 0.3232$$

Using $\frac{d'}{d} = 0.15$ read the mechanical steel ratio from uniaxial interaction chart for $V_{sd} = 0.745$ $\mu_{sd} = 0.3232$ $\omega = 0.79$ $A = -\frac{\omega f_{cd}bd}{\omega} - \frac{0.79 * 14.166 * 300 * 300}{\omega} - 2518 125 mm^2$

$$A_{s,tot} = \frac{\omega_{fcd} bu}{f_{yd}} = \frac{0.79 \times 14.100 \times 300 \times 300}{400} = 2518.125 \, mm$$
$$A = \frac{A_{s,tot}}{2} = \frac{2518.125}{2} = 1259.0625 \, mm^2$$

Check with maximum and minimum reinforcement limit

$$A_{s,min} = max \begin{cases} \frac{0.1 N_{ED}}{f_{yd}} = 237.5 \ mm^2 & OK! \\ 0.002A_c \end{cases}$$
$$A_{s,max} = 0.08 \ A_c = 0.08 \ * \ 300 \ * \ 300 = 7200 \ mm^2 \qquad OK!$$

Using Ø20 provide 4Ø20 on each face

Step 7- Detailing



Example 4.5 [Column]

Design the braced column if it is subjected to the following loading .the column has total length of 6 m. L_e =0.7L

Use C25/30 and S460

If the column is slender .compute the total design moment using

- a) Nominal curvature
- b) Nominal stiffness

Use 400*400 mm section $\frac{d'}{d} = 0.1$ use $\phi(\infty, t_o) = 2$



Solution

Step 1. Material data

$$f_{cd} = \frac{0.85 * 25}{1.5} = 14.16667 mpa$$

$$f_{yd} = \frac{460}{1.15} = 400 mpa$$

$$E_{cm} = 30 Gpa$$

$$E_s = 200 Gpa$$

Step 2- Check slenderness limit

$$\lambda_{lim} = \frac{20ABC}{\sqrt{n}} \quad take \ A = 0.7 \quad B = 1.1 \quad C = 1.7 - r_m$$

where
$$r_m = \frac{m_{01}}{m_{02}} = \frac{140}{140} = 1$$
 $C = 1.7 - (1) = 0.7$

$$n = \frac{N_{ed}}{A_c f_{cd}} = \frac{1650 * 10^3}{14.1667 * 400 * 400} = 0.72794$$
$$\lambda_{lim} = \frac{20 * 0.7 * 1.1 * 0.7}{\sqrt{0.72794}} = 12.63487$$

Step 3-slenderess

$$\begin{split} \lambda &= \frac{l_o}{i} \quad i = \sqrt{\frac{I}{A}} = \sqrt{\frac{213333333}{160000}} = 115.470 \ mm \\ \lambda &= \frac{0.7 * 6000}{115.470} = 36.373 \\ \lambda &> \lambda_{lim} \quad Slender \ column \ consider \ second \ order \ effect \end{split}$$

$$e_a = \frac{l_o}{400} = \frac{4200}{400} = 10.5 \, mm$$

Step 5- Equivalent first order eccentricity

$$e_{e} = max \begin{cases} 0.6e_{02} + 0.4 \ e_{01} \\ 0.4e_{02} \end{cases}$$

$$e_{02} = \frac{M_{02}}{N_{sd}} = \frac{140 \ * \ 10^{6}}{1650 \ * \ 10^{3}} = 84.848 \ mm \\ e_{01} = \frac{M_{01}}{N_{sd}} = \frac{140 \ * \ 10^{6}}{1650 \ * \ 10^{3}} = 84.848 \ mm \\ e_{e} = max \begin{cases} 0.6e_{02} + 0.4 \ e_{01} \\ 0.4e_{02} \end{cases} = 84.848 \ mm \end{cases}$$

Step 6- Calculate the second order moment

a) Using Nominal curvature method

$$e_{2} = \frac{1}{r} l_{o}^{2} / C \qquad C = 10 \quad For \ constant \ cross - section$$

$$\frac{1}{r} = K_{r} K_{\emptyset} \frac{1}{r_{o}} \qquad K_{\emptyset} = 1 + \beta \emptyset_{eff}$$

$$\emptyset_{eff} = 0 \quad if \quad \begin{cases} \emptyset(\infty, t_{o}) \leq 2 \quad OK \\ \lambda \leq 75 \quad OK \\ \frac{M_{oed}}{N_{ed}} \geq h \quad Not \ OK \end{cases}$$

$$\phi_{eff} = \phi(\infty, t_o) \frac{M_{oeqp}}{M_{oed}}$$

where $M_{oeqp} = first \text{ order moment in } quasi - permanent load(SLS)$ $M_{oed} = First \text{ order moment in } design load combination (ULS)$

$$\begin{split} & \phi_{eff} = \phi(\infty, t_o) \frac{M_{oeqp}}{M_{oed}} = 2 * \frac{70}{140} = 1 \\ & \beta = 0.35 + \frac{f_{ck}}{200} - \frac{3}{150} = 0.35 + \frac{25}{200} - \frac{36.375}{150} = 0.2325 \\ & K_{\phi} = 1 + \beta \phi_{eff} = 1 + 0.2325 * 1 = 1.2325 \\ & \frac{1}{r_o} = \frac{\varepsilon_{yd}}{0.45d} = \frac{2 * 10^{-3}}{360} = 123456 * 10^{-5} \qquad d = effective \ depth \\ & K_r = \frac{(n_u - n)}{(n_u - n_{bal})} \quad n_u = 1 + \omega \quad n = \frac{N_{ed}}{A_c f_{cd}} = \frac{1650 * 10^3}{14.166 * 400^2} \\ & = 0.72974 \quad n_{bal} = 0.4 \end{split}$$

For first iteration take $e_2 = 0$

 $e_{tot} = e_o + e_e + e_2 = 10.5 + 84.848 + 0 = 95.348 mm$ $N_{sd} = 1650 KN$ $M_{sd} = N_{sd} * e_{tot} = 157.3242 KNm$

$$v_{sd} = \frac{N_{sd}}{f_{cd}bh} = \frac{1650 * 10^3}{14.1667 * 400 * 400} = 0.72794$$

$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{157.3242 * 10^6}{14.1667 * 400 * 400^2} = 0.17352$$

Using $v_{sd} = 0.72794$ $\mu_{sd} = 0.1752$ $\frac{d'}{d} = 0.1$ $\omega = 0.25$
So

$$(n - n) = (1.25 - 0.72794)$$

$$n_u = 1 + \omega = 1.25 \quad K_r = \frac{(n_u - n)}{(n_u - n_{bal})} = \frac{(1.25 - 0.72794)}{(1.25 - 0.4)} = 0.614$$

$$\frac{1}{r} = 0.614 * 1.2325 * 1.23456 * 10^{-5} = 9.3456 * 10^{-6}$$

$$e_2 = \frac{1}{r} \frac{4200^2}{10} = 16.485 \, mm$$

For Second iteration take $e_2 = 16.485 \ mm$

 $e_{tot} = e_o + e_e + e_2 = 10.5 + 84.848 + 16.485 = 111.833 mm$ $N_{sd} = 1650 KN$ $M_{sd} = N_{sd} * e_{tot} = 184.525 KNm$ Using $v_{sd} = 0.72794$ and $\mu_{sd} = 0.2035$ $\omega = 0.35$

$$n_u = 1 + \omega = 1.35$$
 $K_r = \frac{(n_u - n)}{(n_u - n_{bal})} = \frac{(1.35 - 0.72794)}{(1.35 - 0.4)} = 0.6548$

$$\frac{1}{r} = 0.6548 * 1.2325 * 1.23456 * 10^{-5} = 9.9634 * 10^{-6}$$
$$e_2 = \frac{1}{r} \frac{4200^2}{10} = 17.575 \, mm$$

For their iteration take $e_2 = 17.575 \ mm$

$$e_{tot} = e_o + e_e + e_2 = 10.5 + 84.848 + 17.575 = 112.923 mm$$

 $N_{sd} = 1650 KN$ $M_{sd} = N_{sd} * e_{tot} = 186.323 KNm$
Using $v_{sd} = 0.72794$ and $\mu_{sd} = 0.2055$ $\omega = 0.35$

The iteration **converges** with similar mechanical steel ratio $\,\omega=0.35$.

$$A_{s,tot} = \frac{\omega f_{cd} bd}{f_{yd}} = \frac{0.35 * 14.166 * 400 * 400}{400} = 1983.33 \ mm^2$$
$$A = \frac{A_{s,tot}}{2} = \frac{1983.33}{2} = 991.666 \ mm^2$$

Check with maximum and minimum reinforcement limit

$$A_{s,min} = max \begin{cases} \frac{0.1 N_{ED}}{f_{yd}} = 412.5 \ mm^2 \quad OK! \\ 0.002A_c \end{cases}$$
$$A_{s,max} = 0.08 \ A_c = 0.08 \ * \ 400 \ * \ 400 = 12800 \ mm^2 \qquad OK!$$

Using Ø16 provide 5Ø16 on each face



b) Using Nominal stiffness method

$$EI = K_c E_{cd} I_c + K_s E_s I_s$$
$$E_{cd} = \frac{E_{cm}}{\gamma_{ce}} = \frac{30}{1.2} = 25 \ Gpa$$

Initially let us assume $ho \geq 0.01$ to use the simplified method

$$K_{s} = 0 \quad K_{c} = \frac{0.3}{1 + 0.5} \phi_{eff}$$

$$\phi_{eff} = 1 \quad K_{c} = 0.2$$

$$I_{c} = \frac{bh^{3}}{12} = 2133333333 mm^{4}$$

$$EI = K_{c}E_{cd}I_{c} + K_{s}E_{s}I_{s} = 0.2 * 25 * 10^{3} * 2133333333 = 1.06667 * 10^{13}$$

$$N_{b} = \frac{\Pi^{2}EI}{l_{o}^{2}} = \frac{\Pi^{2} * 1.06667 * 10^{13}}{4200^{2}} = 5968.01475 KN$$

 N_b = Buckling load based on nominal stiffness

Design moment
$$M_{ed} = M_{oed} \left[1 + \frac{\beta}{\binom{N_b}{N_{ed}} - 1}} \right]$$

 $\beta = \frac{\Pi^2}{c_o}$
 $c_o = 8$
 $M_{oed} = 140 \text{ KNm}$
 $M_{ed} = M_{oed} \left[1 + \frac{\beta}{\binom{N_b}{N_{ed}} - 1}} \right] = 140 * \left[1 + \frac{1.2337}{\binom{5968.01475}{1650} - 1} \right]$
 $= 205.999 \text{ KNm}$

For first iteration neglecting accidental eccentricity

$$v_{sd} = \frac{N_{sd}}{f_{cd}bh} = \frac{1650 * 10^3}{14.1667 * 400 * 400} = 0.72794$$

$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{205.999 * 10^6}{14.1667 * 400 * 400^2} = 0.2272$$

Using $v_{sd} = 0.72794$ $\mu_{sd} = 0.2272$ $\frac{d'}{d} = 0.1$ $\omega = 0.4$

$$\begin{split} A_{s,tot} &= \frac{\omega f_{cd} b d}{f_{yd}} = \frac{0.4 * 14.166 * 400 * 400}{400} = 2266.666 \ mm^2 \\ A &= \frac{A_{s,tot}}{2} = \frac{2266.666}{2} = 1133.333 \ mm^2 \\ \rho &= \frac{A_s}{bd} = \frac{1133.333}{400 * 360} = 7.870 * 10^{-3} \quad \rho < 0.01 \end{split}$$

We cannot use the simplified method

for
$$\rho > 0.002$$
 $K_s = 1$

$$K_{c} = \frac{K_{1}K_{2}}{(1 + \phi_{eff})}$$

$$K_{1} = \sqrt{\frac{f_{ck}}{20}} = 1.11803$$

$$K_{2} = n\frac{\lambda}{170} \le 0.2$$

$$n = \frac{N_{ed}}{A_{c}f_{cd}} = 0.72794$$

$$K_{2} = 0.2 \quad So \qquad K_{c} = 0.111803$$

$$I_c = \frac{bh^3}{12} = 2133333333 mm^4$$
$$I_s = 2 * [1133.333(360 - 200)^2] = 58026666.67 mm^2$$

$$EI = K_c E_{cd} I_c + K_s E_s I_s$$

= 0.111803 * 25 * 10³ * 2133333333 + 1 * 200 * 10³
* 58026666.6

$$EI = 1.756816 * 10^{13}$$

$$N_{b} = \frac{\Pi^{2} EI}{l_{o}^{2}} = \frac{\Pi^{2} * 1.756816 * 10^{13}}{4200^{2}} = 9829.40982 \, KN$$
$$M_{ed} = M_{oed} \left[1 + \frac{\beta}{\binom{N_{b}}{N_{ed}} - 1} \right] = 140 * \left[1 + \frac{1.2337}{\binom{9829.40982}{1650} - 1} \right]$$
$$= 174.842 \, KNm$$

Design moment including accidental eccentricity is given by

$$M = 174.842 + 1650 * \left(\frac{10.5}{1000}\right) = 192.167 \, KNM$$

$$v_{sd} = \frac{N_{sd}}{f_{cd}bh} = \frac{1650 * 10^3}{14.1667 * 400 * 400} = 0.72794$$

$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{192.167 * 10^6}{14.1667 * 400 * 400^2} = 0.212$$

Using $v_{sd} = 0.72794$ $\mu_{sd} = 0.212$ $\frac{d'}{d} = 0.1$ $\omega = 0.35$

$$A_{s,tot} = \frac{\omega f_{cd} bd}{f_{yd}} = \frac{0.35 * 14.166 * 400 * 400}{400} = 1983.333 \ mm^2$$
$$A = \frac{A_{s,tot}}{2} = \frac{1983.33}{2} = 991.666 \ mm^2$$

$$\rho = \frac{A_s}{bd} = \frac{991.666}{400 * 360} = 6.886 * 10^{-3} \quad \rho > 0.002$$

Second Iteration

$$\begin{split} I_c &= \frac{bh^3}{12} = 2133333333 \ mm^4 \\ I_s &= 2 * \left[991.666(360 - 200)^2 \right] = 50773333.33 \ mm^2 \\ EI &= K_c E_{cd} I_c + K_s E_s I_s \\ &= 0.111803 * 25 * 10^3 * 2133333333 + 1 * 200 * 10^3 \\ &* 50773333.3 \end{split}$$
$$EI &= 1.611749 * 10^{13} \end{split}$$

$$N_{b} = \frac{\Pi^{2} EI}{l_{o}^{2}} = \frac{\Pi^{2} * 1.611749 * 10^{13}}{4200^{2}} = 9017.7598 \, KN$$
$$M_{ed} = M_{oed} \left[1 + \frac{\beta}{\binom{N_{b}}{N_{ed}} - 1} \right] = 140 * \left[1 + \frac{1.2337}{\binom{9017.7598}{1650} - 1} \right]$$
$$= 178.679 \, KNm$$

Design moment including accidental eccentricity is given by

 $M = 178.679 + 1650 * \left(\frac{10.5}{1000}\right) = 196.004 \text{ KNM}$

$$v_{sd} = \frac{N_{sd}}{f_{cd}bh} = \frac{1650 * 10^3}{14.1667 * 400 * 400} = 0.72794$$

$$\mu_{sd} = \frac{m_{sd}}{f_{cd}bd^2} = \frac{196.004 * 10^6}{14.1667 * 400 * 400^2} = 0.216$$

Using $v_{sd} = 0.72794$ $\mu_{sd} = 0.216$ $\frac{d'}{d} = 0.1$ $\omega = 0.35$

The iteration converges with similar mechanical steel ratio $\,\omega=0.35$.

$$A_{s,tot} = \frac{\omega f_{cd} b d}{f_{yd}} = \frac{0.35 * 14.166 * 400 * 400}{400} = 1983.33 \ mm^2$$
$$A = \frac{A_{s,tot}}{2} = \frac{1983.33}{2} = 991.666 \ mm^2$$

Check with maximum and minimum reinforcement limit

$$A_{s,min} = max \begin{cases} \frac{0.1 N_{ED}}{f_{yd}} = 412.5 \ mm^2 & OK! \\ 0.002A_c \end{cases}$$

$$A_{s,max} = 0.08 A_c = 0.08 * 400 * 400 = 12800 mm^2$$
 OK!



Step 7- Detailing



400mm